



Studienämnden Kf / Kb

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TRANSPORTPROCESSER

HÄRLEDNINGAR

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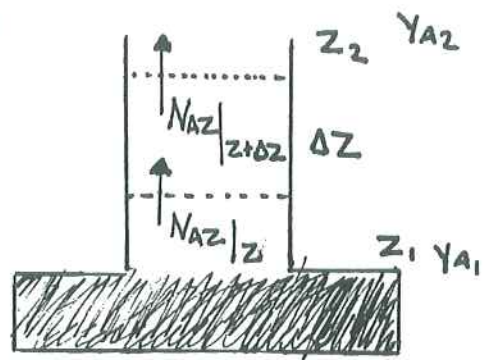
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Arnoldcell

$$N_{AZ} = -c D_{AB} \frac{dY_A}{dz} + Y_A (N_{Az} + N_{Bz})$$

Stagnant gas: $N_{Bz} = 0$

$$\Rightarrow N_{AZ} = -\frac{c D_{AB}}{1 - Y_A} \frac{dY_A}{dz}$$



$$\frac{dN_{AZ}}{dz} = 0$$

$$\Rightarrow \frac{d}{dz} \left(-\frac{c D_{AB}}{1 - Y_A} \frac{dY_A}{dz} \right) = 0 \Rightarrow$$

$$\frac{c D_{AB}}{1 - Y_A} \frac{dY_A}{dz} + C_1 = 0$$

$$C_1 dz = -\frac{1}{1 - Y_A} dY_A \Rightarrow C_1 z = \ln(1 - Y_A) + C_2$$

Gränser: $z_1 \rightarrow z_2$ $Y_{A1} \rightarrow Y_{A2}$

$$\Rightarrow \frac{(1 - Y_{A2})}{(1 - Y_{A1})} = \left(\frac{1 - Y_{A2}}{1 - Y_{A1}} \right)^{\left(\frac{z - z_1}{z_2 - z_1} \right)}$$



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Hastighetsprofil vid fullt utvecklad laminär strömning nedför ett lutande plan

Rörelsemängdsbalans: $\sum F_x = \iint_{CS} v_x \rho (\mathbf{v} \cdot \mathbf{n}) dA + \frac{d}{dt} \iiint_{CV} \rho v_x dV$

(1) $in = ut$

(2) fullt utvecklad strömning

$$\sum F_x = \underbrace{\rho \Delta y \Big|_x - \rho \Delta y \Big|_{x+\Delta x}}_{(2)} + \tau_{yx} \Delta x \Big|_y - \tau_{yx} \Delta x \Big|_{y+\Delta y} + \rho g \Delta x \Delta y \sin \theta$$

Dividera med $\Delta x \Delta y \Rightarrow -\frac{d\tau_{yx}}{dy} - \rho g \sin \theta = 0$

Integration $\Rightarrow \tau_{yx} = C_1 - \rho g y \sin \theta$

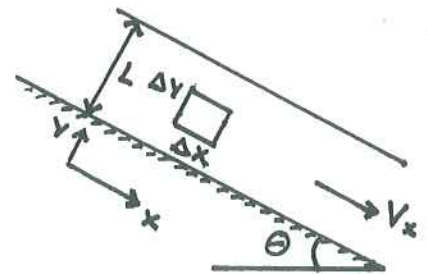
$\tau_{yx}(L) = 0 \Rightarrow C_1 = \rho g L \sin \theta$

$\tau_{yx} = \rho g (L - y) \sin \theta = \mu \frac{dv_x}{dy}$

Integration $\Rightarrow v_x = \frac{\rho g}{\mu} (Ly - \frac{y^2}{2}) \sin \theta + C_2$

$v_x(0) = 0 \Rightarrow C_2 = 0$

$$v_x = \frac{\rho g}{\mu} (Ly - \frac{y^2}{2}) \sin \theta$$





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Hastighetsprofil i rör med axel



rörelsemängdsbalans: $\Sigma F_x = \iint_{CS} \rho v_x (\mathbf{v} \cdot \mathbf{n}) dA + \frac{d}{dt} \iiint_{CV} \rho v_x dV$

(1) in = ut

(2) fullt utvecklade strömning

Endast friktionsförluster \Rightarrow

$$\Sigma F_x = \tau_{rx} (2\pi r \Delta x) \Big|_{r+\Delta r} - \tau_{rx} (2\pi r \Delta x) \Big|_r = 0$$

Dividera med $2\pi \Delta x \Delta r$

$$\Rightarrow \frac{d}{dr} (r \tau_{rx}) = 0 = \frac{d}{dr} (r \mu \frac{dv_x}{dr})$$

Integration $\Rightarrow r \mu \frac{dv_x}{dr} + C_1 = 0 \Rightarrow dv_x = -\frac{C_1}{\mu} \frac{dr}{r}$

Integration $\Rightarrow v_x = -\frac{C_1}{\mu} \ln(r) + C_2$

$v_x(R_2) = 0 \Rightarrow C_2 = \frac{C_1}{\mu} \ln(R_2)$

$v_x(R_1) = v \Rightarrow v = \frac{C_1}{\mu} \ln(R_1) - \frac{C_1}{\mu} \ln(R_2) = \frac{C_1}{\mu} \left(\ln \frac{R_1}{R_2} \right)$

$$\Rightarrow \frac{C_1}{\mu} = \frac{v}{\ln \frac{R_1}{R_2}}$$

$$\Rightarrow v_x(r) = v \frac{\ln(r/R_2)}{\ln(R_1/R_2)}$$

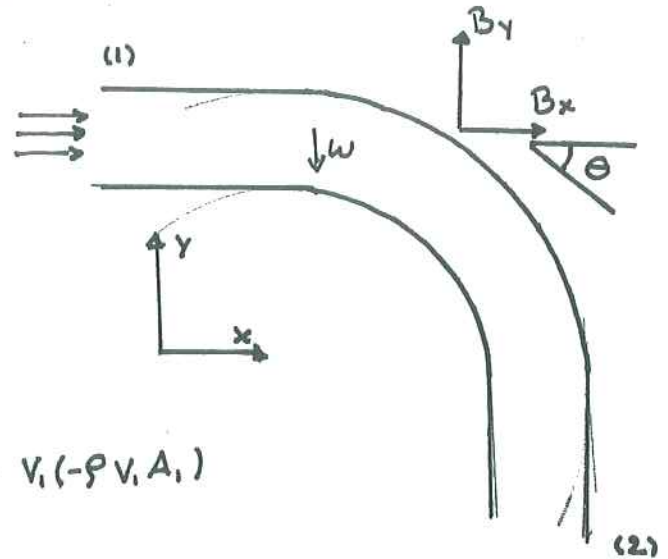


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kraftkomponenter i en rörböj

$$\sum F_x = P_1 A_1 - P_2 A_2 \cos \theta + B_x$$

$$\sum F_y = P_2 A_2 \sin \theta - \omega + B_y$$



$$\iint_{CS} v_x \rho (\mathbf{v} \cdot \mathbf{n}) dA = (v_2 \cos \theta) (\rho v_2 A_2) + v_1 (-\rho v_1 A_1)$$

$$\iint_{CS} v_y \rho (\mathbf{v} \cdot \mathbf{n}) dA = (-v_2 \sin \theta) (\rho v_2 A_2)$$

fullt utvecklade strömning $\Rightarrow \frac{d}{dt} \iiint_{CV} = 0$

$$P_1 A_1 - P_2 A_2 \cos \theta + B_x = (v_2 \cos \theta) (\rho v_2 A_2) + v_1 (-\rho v_1 A_1)$$

$$P_2 A_2 \sin \theta - \omega + B_y = (-v_2 \sin \theta) (\rho v_2 A_2)$$

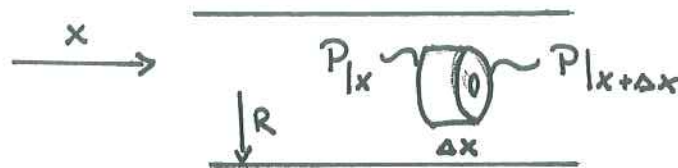
$$B_x = v_2^2 \rho A_2 \cos \theta - v_1^2 \rho A_1 - P_1 A_1 + P_2 A_2 \cos \theta$$

$$B_y = -v_2^2 \rho A_2 \sin \theta - P_2 A_2 \sin \theta + \omega$$



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Hastighetsprofil vid fullt utvecklade, horisontella, laminära förströmning



Rörelsemängdsbalans: $\sum F_x = \iint_{CS} \rho v_x (v \cdot n) dA + \frac{d}{dt} \iiint_{CV} \rho v_x dV$

(1) $in = ut$

(2) fullt utvecklade strömning

$$\sum F_x = P 2\pi r \Delta r \Big|_x - P 2\pi r \Delta r \Big|_{x+\Delta x} + \tau_{rx} 2\pi r \Delta x \Big|_{r+\Delta r} - \tau_{rx} 2\pi r \Delta x \Big|_r$$

dividera med $2\pi r \Delta r \Delta x$

$$\Rightarrow -r \frac{dP}{dx} + \frac{d}{dr} (r \tau_{rx}) = 0$$

integration $\Rightarrow \tau_{rx} r = \frac{dP}{dx} \frac{r^2}{2} + C_1 \Rightarrow \tau_{rx} = \frac{dP}{dx} \frac{r}{2} + \frac{C_1}{r}$

C_1 måste vara noll

för att undvika att $\frac{C_1}{r} \rightarrow \infty$

$$\tau_{rx} = \mu \frac{dv_x}{dr} \Rightarrow \mu \frac{dv_x}{dr} = \frac{dP}{dx} \frac{r}{2}$$

integration $\Rightarrow v_x = \frac{dP}{dx} \frac{1}{4\mu} \frac{r^2}{2} + C_2$

$$v_x(R) = 0 \Rightarrow C_2 = \frac{1}{4\mu} \frac{dP}{dx} R^2$$

$$\Rightarrow v_x = \frac{dP}{dx} \frac{1}{4\mu} (r^2 - R^2)$$



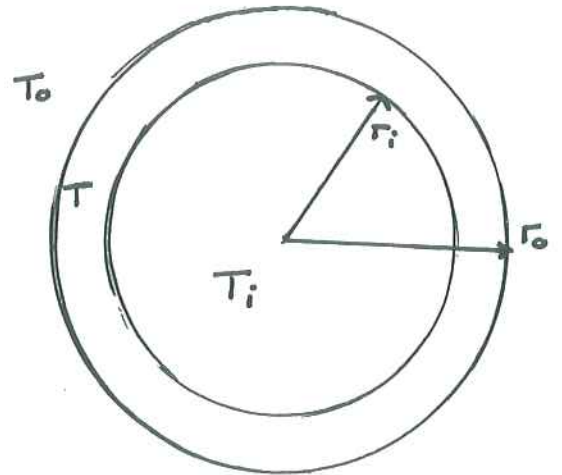
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Medelarea för stationär värmeledning genom cylindriskt skal

$$q = \frac{k \bar{A}}{r_o - r_i} \Delta T$$

$$q = -k A \frac{dT}{dr} \quad A = 2\pi r L$$

integrering $r_i \rightarrow r_o$ $T_i \rightarrow T_o$ $q = \text{konstant}$



$$\ln\left(\frac{r_o}{r_i}\right) = -\frac{2\pi L k}{q} (T_o - T_i)$$

$$\Rightarrow q = \frac{2\pi L k (T_i - T_o)}{\ln(r_o/r_i)} = \frac{2\pi L k (T_i - T_o) (r_o - r_i)}{\ln(r_o/r_i) (r_o - r_i)} = \frac{k \bar{A}}{r_o - r_i} \Delta T$$

$$\Rightarrow \bar{A} = \frac{2\pi L (r_o - r_i)}{\ln(r_o/r_i)}$$

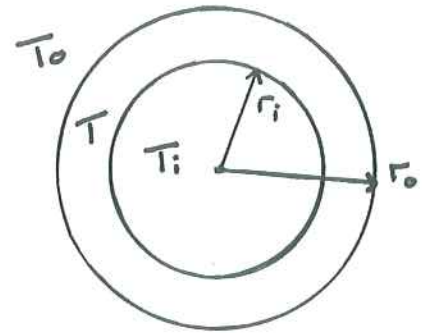


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Temperaturprofil i cylindriskt skal

$$q = kA \frac{dT}{dr} \quad A = 2\pi r L$$

$$\Rightarrow \frac{q}{L} = \frac{2\pi k}{\ln(r_0/r_i)} (T_i - T_0)$$



Accumulation = in - ut + producerat

$$\Rightarrow \frac{dq}{dr} = 0$$

$$\Rightarrow \frac{d}{dr} \left(\frac{r dT}{dr} \right) = 0 \quad \text{enligt definition}$$

$$\text{integration} \Rightarrow r \frac{dT}{dr} = -C_1 \Rightarrow T = -C_1 \ln(r) + C_2$$

$$T(r_i) = T_i \Rightarrow C_2 = T_i + C_1 \ln(r_i)$$

$$T(r_0) = T_0 \Rightarrow C_1 = \frac{T_0 - T_i}{\ln(r_i/r_0)}$$

$$\Rightarrow T = \frac{T_0 - T_i}{\ln(r_0/r_i)} \ln(r) + T_i + \frac{T_0 - T_i}{\ln(r_0/r_i)} \ln(r_i)$$

$$\Rightarrow T = T_i + \frac{T_0 - T_i}{\ln(r_0/r_i)} \ln(r/r_i)$$